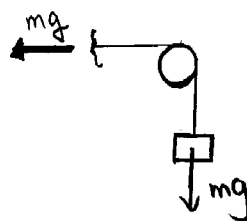
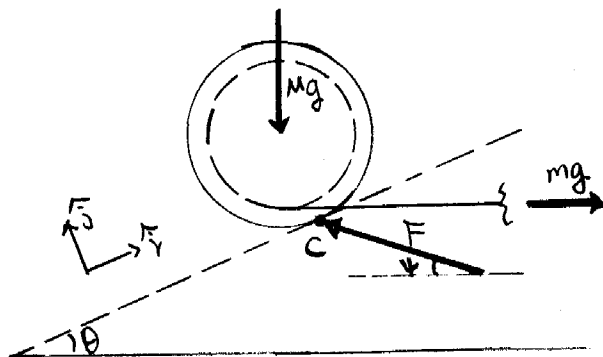


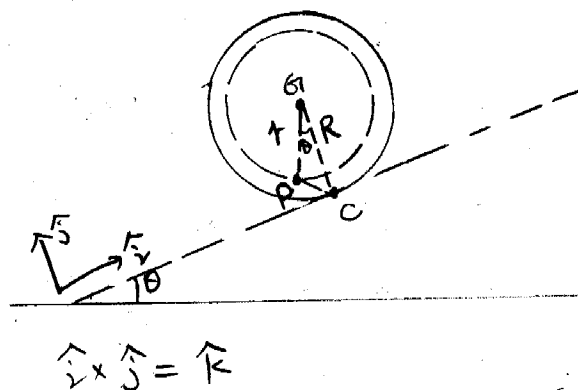
TAM 203 solns by Jing Shi
 HW due Tues March 27, 2007
 Problem: 4.69 4.72 4.97 12.16 12.18

4.69 Soln:

FBD:



Geometry



(1) The system is at rest \Rightarrow statics

$$\sum \vec{M}_C = \{ \vec{r}_{G/C} \times (-Mg\hat{e}_2) + \vec{r}_{P/C} \times (mg\hat{e}_1) \} \cdot \hat{k} = 0$$

$$\vec{r}_{G/C} = R\sin\theta\hat{e}_1 - R\cos\theta\hat{e}_2$$

$$\vec{r}_{P/C} = R\sin\theta\hat{e}_1 - (R\cos\theta - t)\hat{e}_2$$

$$M_C = \{ (R\sin\theta\hat{e}_1 - R\cos\theta\hat{e}_2) \times (-Mg\hat{e}_2) + (R\sin\theta\hat{e}_1 - (R\cos\theta - t)\hat{e}_2) \times (mg\hat{e}_1) \} \cdot \hat{k} = 0$$

$$M_C = \{ -MgR\sin\theta\hat{k} + (R\cos\theta - t)mg\hat{k} \} \cdot \hat{k}$$

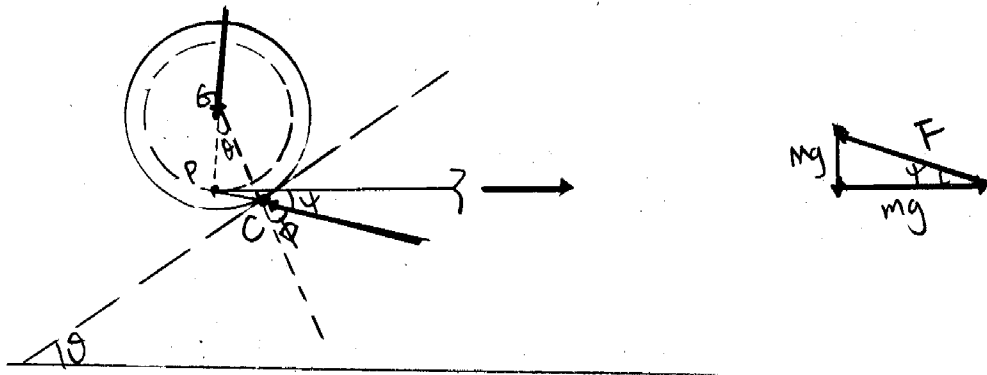
$$= -MgR\sin\theta + (R\cos\theta - t)mg = 0$$

$$\frac{m}{M} = \frac{\sin\theta}{\cos\theta - \frac{1}{2}}$$

$$\begin{aligned} \hat{e}_1 \times \hat{e}_2 &= \hat{k} \\ \hat{e}_1 \times \hat{e}_1 &= 0 \\ \hat{e}_2 \times \hat{e}_2 &= 0 \\ \hat{k} \cdot \hat{k} &= 1 \end{aligned}$$

a) alternative solution

The reel is a three force body so the direction of force \vec{F} at C is determined: its line of action must pass through the intersection of line of action of other two forces.



From triangle ΔGPC

$$\frac{R}{\sin(\pi - \phi - \theta)} = \frac{|\vec{F}_{PC}|}{\sin \theta}$$

$$\sin(\pi - \phi - \theta) = \frac{R \sin \theta}{|\vec{F}_{PC}|}$$

$$\pi - \phi - \theta = \psi + \frac{\pi}{2}$$

$$\therefore \psi = \frac{\pi}{2} - \phi - \theta$$

$$\cos \psi = \cos\left(\frac{\pi}{2} - \phi - \theta\right) = \sin(\phi + \theta) = \sin(\pi - \phi - \theta)$$

$$\cos \psi = \frac{R \sin \theta}{|\vec{F}_{PC}|}$$

$$\begin{aligned} \frac{M}{m} = \tan \psi &= \frac{\sin \psi}{\cos \psi} \\ &= \frac{\sqrt{1 - \frac{R \sin \theta}{|\vec{F}_{PC}|}}}{\frac{R \sin \theta}{|\vec{F}_{PC}|}} \\ &= \frac{\sqrt{|\vec{F}_{PC}|^2 - R^2 \sin^2 \theta}}{R \sin \theta} \end{aligned}$$

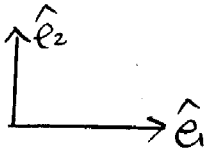
$$\left. \begin{aligned} |\vec{F}_{PC}|^2 &= R^2 + r^2 - 2rR \cos \theta \\ r &= \frac{1}{2}R \end{aligned} \right\} \Rightarrow \begin{aligned} &= \frac{R \sin \theta}{\sqrt{R^2 \cos^2 \theta + \frac{1}{4}R^2 - R^2 \cos \theta}} \\ &= \frac{\cos \theta - \frac{1}{2}}{\sin \theta} \end{aligned}$$

$$\boxed{\frac{m}{M} = \frac{\sin \theta}{\cos \theta - \frac{1}{2}}}$$

b) The pulley is massless

$$T = mg = \frac{\sin \theta}{\cos \theta - \frac{1}{2}} Mg$$

c)



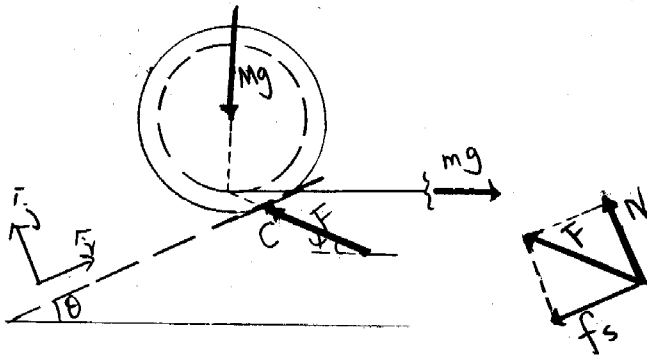
$$\sum F_x = 0 = mg + F_x = 0 \quad F_x = -\frac{\sin \theta}{\cos \theta - \frac{1}{2}} Mg$$

$$\sum F_y = 0 = -Mg + F_y = 0 \quad F_y = Mg$$

$$\vec{F} = -\frac{\sin \theta}{\cos \theta - \frac{1}{2}} Mg \hat{e}_1 + Mg \hat{e}_2$$

$$= Mg \left(-\frac{\sin \theta}{\cos \theta - \frac{1}{2}} \hat{e}_1 + \hat{e}_2 \right)$$

d)



If the wheel is about to slip

$$f_s = \mu N \quad \mu = \frac{f_s}{N}$$

$$\sum \vec{F} = \vec{0}$$

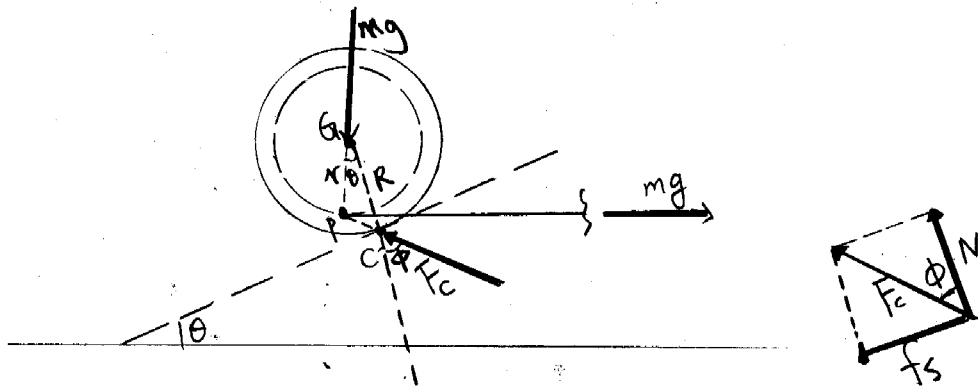
$$-Mg(\sin \theta \hat{i} + \cos \theta \hat{j}) + mg(\cos \theta \hat{i} - \sin \theta \hat{j}) - f_s \hat{i} + N \hat{j} = 0$$

$$f_s = -Mg \sin \theta + mg \cos \theta = Mg \left(-\sin \theta + \frac{\sin \theta \cos \theta}{\cos \theta - \frac{1}{2}} \right)$$

$$N = Mg \cos \theta + mg \sin \theta = Mg \left(\cos \theta + \frac{\sin \theta}{\cos \theta - \frac{1}{2}} \right)$$

$$\mu = \frac{f_s}{N} = \frac{\frac{1}{2} \sin \theta}{1 - \frac{1}{2} \cos \theta}$$

d) alternative solution



$$\vec{F}_c = \vec{N} + \vec{f}_s \quad f_s = \mu N \quad \mu = \frac{f_s}{N} = \tan \phi$$

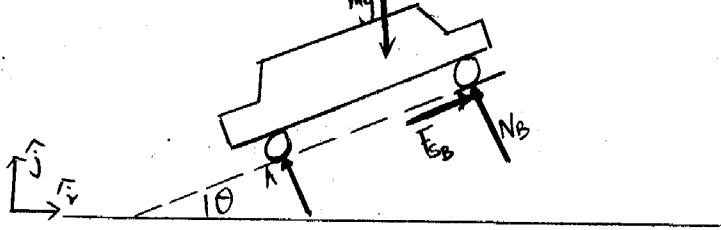
$\tan \phi$ can be calculated by the geometry of triangle ΔGPC

$$\frac{r}{\sin \phi} = \frac{|\vec{F}_c \cos \phi|}{\sin \theta} \Rightarrow \sin \phi = \frac{r \sin \theta}{|\vec{F}_c|}$$

$$\therefore |\vec{F}_c|^2 = r^2 + R^2 - 2rR \cos \theta$$

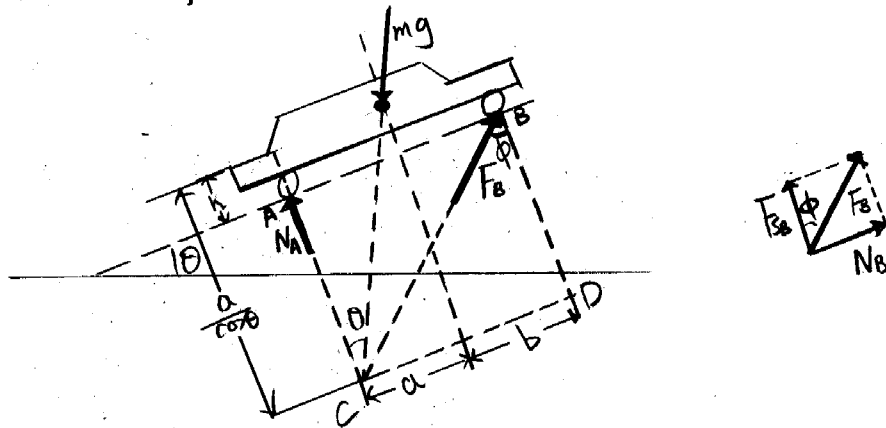
$$\begin{aligned} \therefore \tan \phi &= \frac{\sin \phi}{\cos \phi} \\ &= \frac{\sin \phi}{\sqrt{1 - \sin^2 \phi}} \\ &= \frac{r \sin \theta}{|\vec{F}_c|} \\ &= \frac{r \sin \theta}{\sqrt{r^2 + R^2 - 2rR \cos \theta - r^2 \sin^2 \theta}} \\ r = \frac{1}{2}R &\Rightarrow = \frac{\frac{1}{2} \sin \theta}{\sqrt{\frac{1}{4} + 1 - \cos \theta - \frac{1}{4} \sin^2 \theta}} \\ &= \frac{\frac{1}{2} \sin \theta}{1 - \frac{1}{2} \cos \theta} \end{aligned}$$

4.72 Soln:



Method 1

The car is a three force body, so the direction of the resultant force \vec{F}_B is determined. Its line of action must pass through the intersection of line of action of other two forces.



$$\vec{F}_B = \vec{N}_B + \vec{F}_{sB} \quad F_{sB} = \mu N_B \quad \mu = \frac{F_{sB}}{N_B} = \tan \phi$$

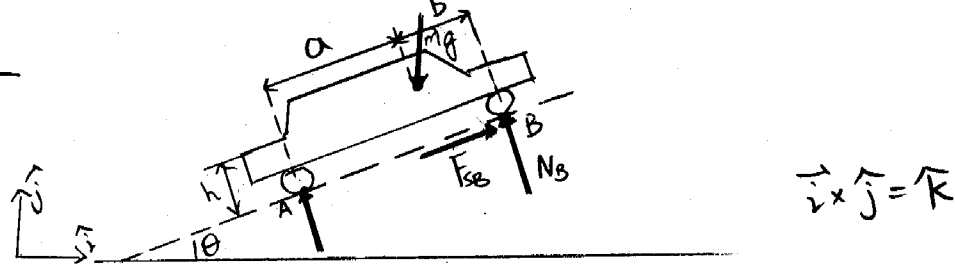
$\tan \phi$ can be calculated by the geometry of triangle ΔCDB

$$|\vec{r}_{CD}| = a+b \quad |\vec{r}_{CB}| = \frac{a}{\cos \theta} - h$$

$$\tan \phi = \frac{|\vec{r}_{CD}|}{|\vec{r}_{CB}|} = \frac{a+b}{\frac{a}{\cos \theta} - h}$$

$$\mu = \frac{(a+b) \sin \theta}{a \cos \theta - h \sin \theta}$$

Method 2



$$\mathcal{J}E_{MA} = \{ \vec{r}_{CA} \times (mg \hat{j}) + \vec{r}_{BA} \times N_B (-\sin\theta \hat{i} + \cos\theta \hat{j}) \} \cdot \hat{k} = 0$$

$$\vec{r}_{CA} = (a \cos\theta - h \sin\theta) \hat{i} + (a \sin\theta + h \cos\theta) \hat{j}$$

$$\vec{r}_{BA} = (a+b) \cos\theta \hat{i} + (a+b) \sin\theta \hat{j}$$

$$E_{MA} = \left\{ \left((a \cos\theta - h \sin\theta) \hat{i} + (a \sin\theta + h \cos\theta) \hat{j} \right) \times (mg \hat{j}) + \right.$$

$$\left. \left((a+b) \cos\theta \hat{i} + (a+b) \sin\theta \hat{j} \right) \times N_B (-\sin\theta \hat{i} + \cos\theta \hat{j}) \right\} \cdot \hat{k}$$

$$= \left\{ (a \cos\theta - h \sin\theta) mg \hat{k} + N_B (a+b) (\sin^2\theta + \cos^2\theta) \hat{k} \right\} \cdot \hat{k}$$

$$= (a \cos\theta - h \sin\theta) mg + N_B (a+b) = 0$$

$$N_B = \frac{mg(h \sin\theta - a \cos\theta)}{a+b}$$

$$\sum \vec{F} = 0$$

$$F_{SB} - mg \sin\theta = 0$$

$$F_{SB} = mg \sin\theta$$

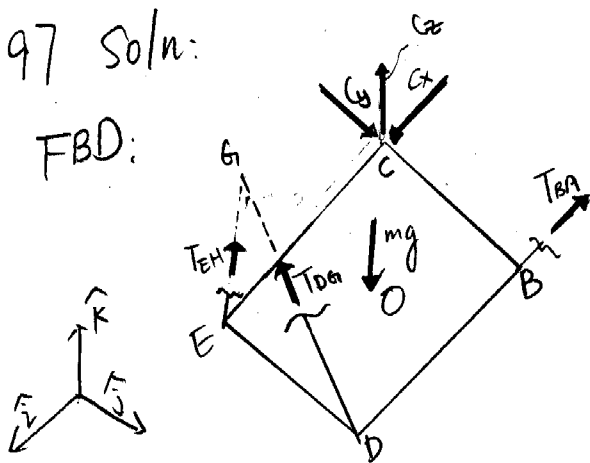
$$N_B \mu = F_{SB} \quad \mu = \frac{F_{SB}}{N_B}$$

$$\mu = \frac{\sin\theta(a+b)}{h \sin\theta - a \cos\theta}$$

$\hat{i} \times \hat{i} = 0$
$\hat{i} \times \hat{j} = \hat{k}$
$\hat{j} \times \hat{j} = 0$

4.97 Sol'n:

a) FBD:



b) see part n) for all possible answers

c) $\sum \vec{F} = 0$

$$\begin{aligned} \sum F_x &= C_x - T_{BA} = 0 \\ \sum F_y &= -T_{DB} \cos \theta + C_y = 0 \\ \sum F_z &= T_{EH} + T_{DB} \sin \theta - mg + C_z = 0 \end{aligned}$$

d) $\sum \vec{M}_C = 0$

$$\vec{r}_{CO} \times \vec{T}_{DB} + \vec{r}_{CB} \times \vec{T}_{BA} + \vec{r}_{CE} \times \vec{T}_{EH} + \vec{r}_{CO} \times (-mg \hat{k}) = 0$$

$$(\vec{i} + \vec{j}) \times T_{DB} (-\cos \theta \vec{j} + \sin \theta \vec{k}) + \vec{j} \times (-T_{BA} \vec{i}) + \vec{i} \times (T_{EH} \vec{k}) + \frac{1}{2}(\vec{i} + \vec{j}) \times (-mg \hat{k}) = 0$$

$$\begin{aligned} \vec{i} \times \vec{i} &= 0 \\ \vec{j} \times \vec{j} &= 0 \\ \vec{i} \times \vec{j} &= \hat{k} \\ \vec{j} \times \vec{k} &= \hat{i} \\ \vec{i} \times \vec{k} &= -\hat{j} \end{aligned}$$

$$\begin{aligned} \sum \vec{M}_C &= -T_{DB} \cos \theta \hat{k} - T_{DB} \sin \theta \hat{j} + T_{DB} \sin \theta \hat{i} + T_{BA} \hat{k} - T_{EH} \hat{j} + \frac{1}{2}mg \hat{j} - \frac{1}{2}mg \hat{i} \\ &= (T_{DB} \sin \theta - \frac{1}{2}mg) \hat{i} + (-T_{DB} \sin \theta - T_{EH} + \frac{1}{2}mg) \hat{j} + (-T_{DB} \cos \theta + T_{BA}) \hat{k} \\ &= 0 \end{aligned}$$

e) The force balance equations are in (c) above
The moment balance equations:

$$\begin{aligned} \sum M_x = 0 &: T_{DB} \sin \theta - \frac{1}{2}mg = 0 \\ \sum M_y = 0 &: -T_{DB} \sin \theta - T_{EH} + \frac{1}{2}mg = 0 \\ \sum M_z = 0 &: -T_{DB} \cos \theta + T_{BA} = 0 \end{aligned}$$

f)

```
% Engr 203 homework solution, produced by Jing Shi
% Due Tues March 27, 2007
% Problem 4.97 f)
m=5;
g=10;
mg=m*g;
theta=pi/4;

%construct the coefficient matrix

A=[1,0,0,0,0,-1;
   0,1,0,0,-cos(theta),0;
   0,0,1,1,sin(theta),0;
   0,0,0,0,sin(theta),0;
   0,0,0,-1,-sin(theta),0;
   0,0,0,0,-cos(theta),1];

b=[0;0;mg;mg/2;-mg/2;0];

% solve for the unknown forces
% X=[Cx;Cy;Cz;T_EH;T_DG;T_BA]

X=A\b;

Results:

X =

    25.0000
    25.0000
    25.0000
         0
    35.3553
    25.0000
```


g) Take moment about CD

$C_x, C_y, C_z, T_{DE}, T_{BA}$ all go through this axis

$$\sum M_{CD} = 0 \quad \vec{T}_{DE} \times \vec{T}_{EH} = 0$$

$$\boxed{T_{EH} = 0}$$

h) Challenge: For how many of the reactions can you find one equation with one unknown?

① T_{EH} (see (g) above)

② T_{BA} : $\sum M_{BA} = 0$ → Take moment about BA

$$\sum M_{BA} = \frac{1}{\sqrt{2}} (\hat{k} + \hat{i}) \cdot \left[\frac{1}{2} (\hat{i} + \hat{j}) \times (-mg \hat{k}) + \hat{j} \times (-T_{BA} \hat{i}) \right]$$

$$= \frac{1}{\sqrt{2}} (\hat{k} + \hat{i}) \cdot \left[+\frac{1}{2} mg \hat{j} - \frac{1}{2} mg \hat{i} + T_{BA} \hat{k} \right]$$

$$= \frac{1}{\sqrt{2}} (T_{BA} - \frac{1}{2} mg) = 0$$

$$\boxed{T_{BA} = \frac{1}{2} mg}$$

③ C_z $\sum M_{ED} = 0$ → Take moment about ED

$$\sum M_{ED} = \hat{j} \cdot \left[-\hat{i} \times (C_z \hat{k} + \frac{1}{2} (-\hat{i} + \hat{j}) \times (-mg \hat{k})) \right]$$

$$= \hat{j} \cdot \left((C_z - \frac{1}{2} mg) \hat{j} - \frac{1}{2} mg \hat{i} \right) = 0$$

$$= C_z - \frac{1}{2} mg = 0$$

$$\boxed{C_z = \frac{1}{2} mg}$$

④ C_y Take moment about G_1 in the \hat{i} direction

$$\left(\sum \vec{M}_{G_1} \right) \cdot \hat{i} = \sum M_{G_1x} = 0$$

$$\sum M_{G_1x} = \hat{i} \cdot \left[-(\hat{k} + \hat{i}) \times (C_y \hat{j}) + \left(-\frac{1}{2} \hat{i} + \frac{1}{2} \hat{j} - \hat{k} \right) \times (-mg \hat{k}) \right]$$

$$= \hat{i} \cdot \left[+C_y \hat{i} - C_y \hat{k} - \frac{1}{2} mg \hat{j} - \frac{1}{2} mg \hat{i} \right]$$

$$= +C_y - \frac{1}{2} mg = 0$$

$$\boxed{C_y = \frac{1}{2} mg}$$

⑤ T_{DG}

$\sum \tau_{CE} = 0$ → Take moment about CE.

$$\begin{aligned}\tau_{CE} &= -\hat{i} \cdot \left[\hat{j} \times (-\hat{j} + \hat{k}) \frac{T_{DG}}{\sqrt{2}} + \left(-\frac{L}{2}\hat{i} + \frac{L}{2}\hat{j} \right) \times (-mg\hat{k}) \right] \\ &= -\hat{i} \cdot \left[+\frac{T_{DG}}{\sqrt{2}}\hat{i} - \frac{1}{2}mg\hat{j} - \frac{1}{2}mg\hat{i} \right] = \frac{-T_{DG}}{\sqrt{2}} + \frac{1}{2}mg = 0\end{aligned}$$

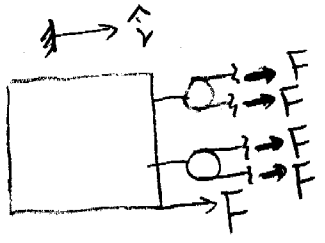
$$\boxed{T_{DG} = \frac{\sqrt{2}}{2}mg}$$

All the answers are the same as the results in part C. We find 5 of 6 using 1 equation with 1 unknown

Note: There is no single eqn. to find C_x

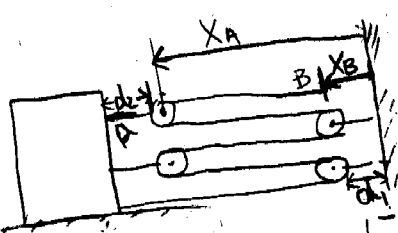
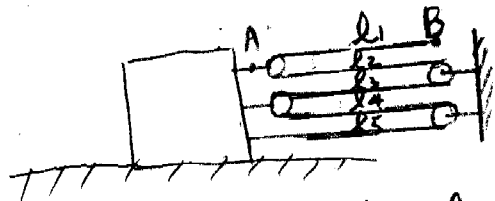
12.16
a)

FBD:



linear momentum balance:
 $5F = ma$

$$a_A = \frac{5F}{M}$$



Note: ignore string wrapped around pulleys

kinematics:

$$l_{tot} = l_1 + l_2 + l_3 + l_4 + l_5 = (x_A - x_B + 4(x_A - d_1) + d_2)$$

$$\vec{a}_A = -\ddot{x}_A(-i)$$

$$\vec{a}_B = \ddot{x}_B(-i)$$

d_1, d_2 are const $\Rightarrow \dot{l}_{tot} = 5\dot{x}_A - \dot{x}_B$, $a_A = -\ddot{x}_A$, $a_B = -\ddot{x}_B$

$$\dot{l}_{tot} = 0 \Rightarrow 5\dot{x}_A = \dot{x}_B \Rightarrow 5a_A = a_B$$

$$a_B = 5a_A = \frac{25F}{m}$$

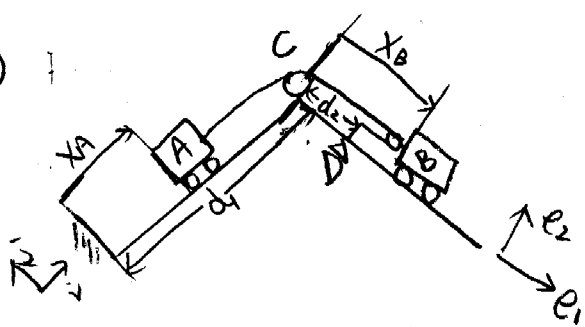
or we can use energy balance

$$F v_B = ma \cdot v_A$$

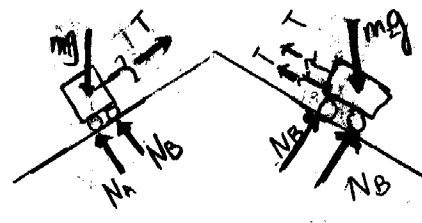
$$\therefore v_B = 5v_A$$

$$a_B = 5a_A = \frac{25F}{m}$$

b)



FBD:



Linear momentum

$$F - mg \sin 30^\circ = m_1 a_A \quad (1)$$

$$mg \sin 60^\circ - 2T = m_2 a_B \quad (2)$$

kinematics:

$$l_{AC} + l_{CB} + l_{BD} = l_{tot} = d_1 - x_A + x_B + x_B - d_2$$

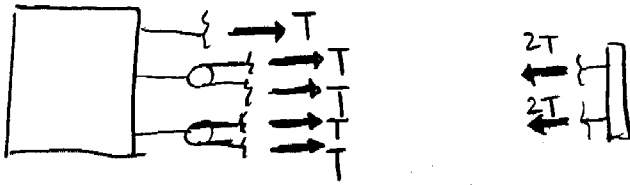
d_1, d_2 are const $\Rightarrow \dot{l}_{tot} = -\dot{x}_A + 2\dot{x}_B = 0$

$$\dot{x}_A = v_A, \dot{x}_B = v_B \Rightarrow a_A = 2a_B \quad (3)$$

$$(1) (2) (3) \Rightarrow a_A = \frac{\frac{5}{2}m_2g - m_1g}{2m_1 + \frac{1}{2}m_2}$$

$$a_B = \frac{\frac{5}{2}m_2g - mg}{4m_1 + m_2}$$

FBD



$$4T = F \quad T = \frac{F}{4}$$

Linear momentum balance $5T = mAa$

$$a_A = \frac{5F}{4m}$$

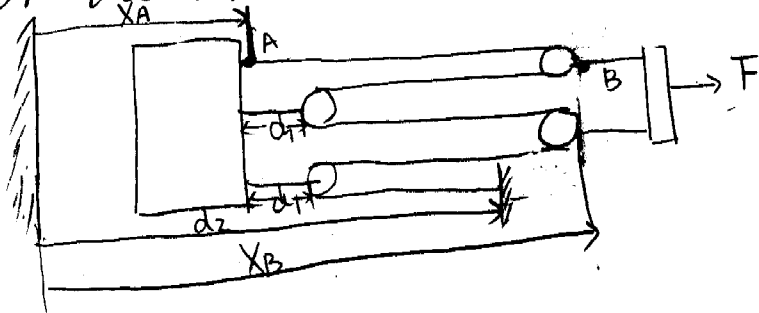
Use Energy balance

$$FV_B = mAaV_A$$

$$V_B = \frac{5}{4}V_A$$

$$a_B = \frac{5}{4}a_A = \frac{25F}{16m}$$

or use kinematics



Ignore the string wrapped around pulleys

$$l_{tot} = X_B - X_A + (X_B - X_A - d_1) \cdot 3 + d_2 - X_A - d_1$$

$$= 4X_B - 5X_A - 4d_1 + d_2$$

d_1, d_2 are const

$$\therefore l_{tot} = 4\dot{X}_B - 5\dot{X}_A \quad \dot{X}_B = \frac{5}{4}\dot{X}_A$$

$$\ddot{X}_B = \frac{5}{4}\ddot{X}_A$$

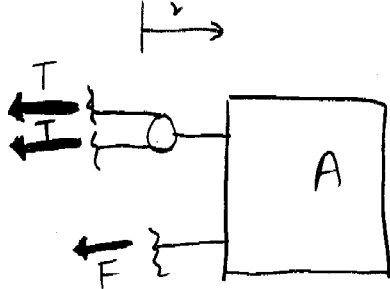
$$\dot{X}_A = a_A \quad \dot{X}_B = a_B$$

$$a_B = \frac{5}{4}a_A = \frac{25F}{16m}$$

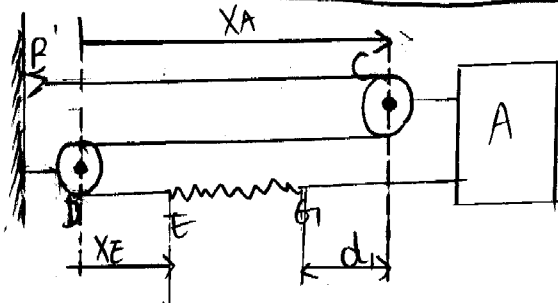
12.28 Soln:

a)

FBD



$$m a_A = -2T - F = -3F$$



Ignore the string wrapped around pulleys.

$$l_{\text{string}} = 2X_A + X_E \quad \Delta l_{\text{string}} = 0 \Rightarrow 2\Delta X_A + \Delta X_E = 0 \quad (1)$$

$$l_{\text{spring}} = X_A - d_1 - X_E \quad \left. \begin{array}{l} d_1 \text{ is const} \\ \Rightarrow \end{array} \right\} \Delta l_{\text{spring}} = \Delta X_A - \Delta X_E \quad (2)$$

$$(1) \cdot (2) \Rightarrow$$

$$\Delta l_{\text{spring}} = 3\Delta X_A$$

$$F = \Delta l_{\text{spring}} \cdot k = 3\Delta X_A \cdot k$$

$$\text{now } \Delta X_A = d$$

$$m a_A = -3F = -3 \cdot 3\Delta X_A \cdot k = -9dk$$

$$a_A = \frac{-9kd}{m}$$

b) The energy is conserved

$$\text{at } \Delta X_A = d \quad E_k = 0 \quad E_p = \frac{1}{2}k(3d)^2 = \frac{9}{2}kd^2$$

$$\Delta X_A = 0 \quad E_p = 0 \quad E_k = \frac{1}{2}mV^2$$

$$\therefore \frac{1}{2}mV^2 = \frac{9}{2}kd^2$$

$$V = 3\sqrt{\frac{k}{m}} d$$